

Name:

Enrolment No:



## UNIVERSITY OF PETROLEUM AND ENERGY STUDIES

End Semester Examination, December 2019

Program Name: B-Tech Mechatronics

Course Name: Mechanical Vibrations

Course Code: MHEG483

Nos. of page(s):

Semester: VII

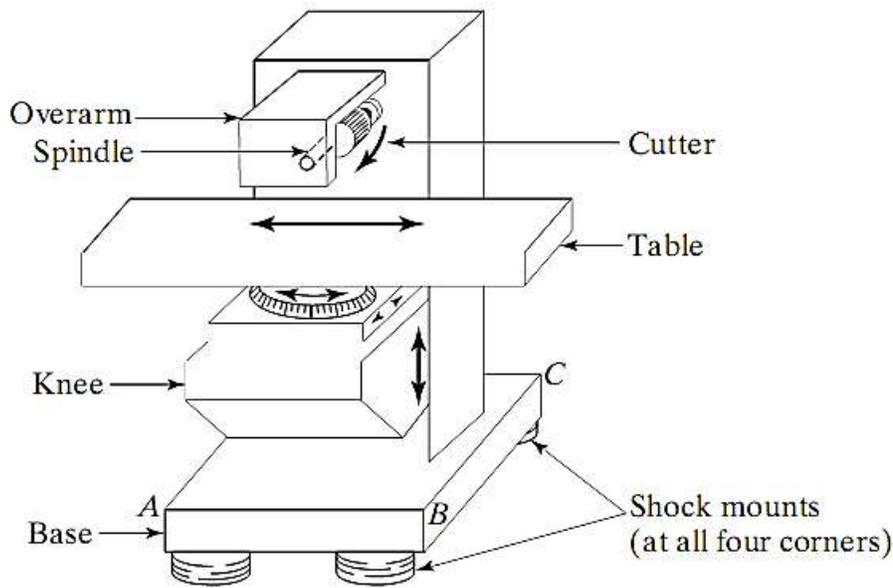
Time: 03 hrs

Max. Marks: 100

**Instructions: If data is insufficient, make relevant assumptions and state the same.**

### SECTION A (Marks-30)

Sl. No.		Marks	CO
Q 1	<p>a) What will be the change in natural frequency of simple pendulum if its length is halved?</p> <p>b) State the most important reason for whirling of shafts to occur.</p> <p>c) The mass matrix of multibody freedom system is always symmetric. State True/False</p> <p>d) Both mass and damping coupling are known as ---</p> <p>e) In a system with rotational imbalance the effect of damping becomes negligibly small at higher speeds. State True/False</p>	5x1 =5	CO1 CO1 CO1 CO1 CO1
Q 2	<p>a) A mass of 10 kg is supported on a spring having a stiffness of 980 N/m and a mass of 5 kg. The damping coefficient is 8 N s/m. Determine the natural frequency of system. Find also the logarithmic decrement and the amplitude after 3 cycles.</p> <p>b) Explain the difference between Dunkerley's method and Stodola method for determining the natural frequencies of a multi body sytem.</p> <p>c) A rotor having a mass of 10 kg is mounted midway on a 0.01m dia shaft supported at the ends by two bearings. The bearing span is 0.5m. Because of certain manufacturing inaccuracies the CG of disc is 0.03 mm away from its geometric cener of rotor. If the system rotates at 3000 rpm find the amplitude of steady state vibrations and the dynamic force transmitted to the bearings. E=200 GPa</p> <p>d) What is meant by static and dynamic coupling? How can you eliminate coupling of the equations of motion?</p> <p>e) Develop a suitable mathematical model for the given system. (Draw the model. No need to derive the EOM)</p>	5x5=25	CO2 CO1 CO3 CO1 CO2 CO3



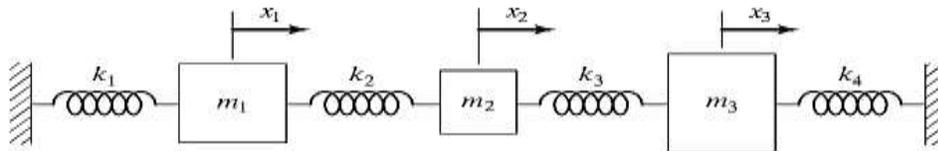
**SECTION B (Marks-30)**

**Q3** The damped natural frequency of a system as obtained from a free vibration test is 9.8 Hz. During the forced vibration test with constant exciting force on the same system, the maximum amplitude of vibration is found to be at 9.6 Hz. Find the damping factor of the system and its natural frequency.

**10**

**CO3**

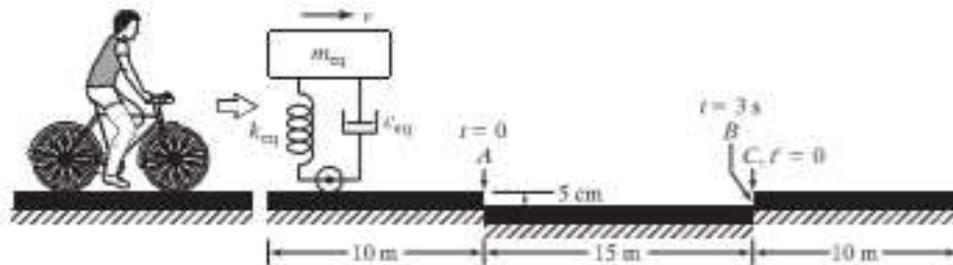
**Q4** Derive the equation of motion (EOM) of the system given below.



**10**

**CO2  
CO3**

**Q5** A boy riding a bicycle can be modeled as a spring-mass-damper system with an equivalent weight, stiffness, and damping constant of 900 N, 55,000 N/m, and 1,500 N-s/m, respectively. The differential setting of the concrete blocks on the road caused the level surface to decrease suddenly, as indicated in Fig. If the speed of the bicycle is 20 km/hr, determine the displacement of the boy in the vertical direction. Assume that the bicycle is free of vertical vibration before encountering the step change in the vertical displacement.



**10**

**CO4  
CO3**

	OR		
Q5	A compound pendulum with mass of rod $M_r$ and mass of bob $M_b$ is oscillating freely on its hinge. If the length of pendulum is $L$ find its frequency of oscillation.	10	C04 C03

**SECTION-C (Marks 40)**

Q6	Using Dunkerley's method find the fundamental natural frequency of the following system.	20	C02
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The diagram shows a horizontal beam supported by two pin supports at the ends. The beam is divided into three equal segments of 1.5 m each. A mass of 100 kg is attached to the first segment, and a mass of 50 kg is attached to the second segment. The beam has a constant flexural rigidity  $EI$  and a weight  $W_g = 10 \text{ kg}$ .

Q7	Find the natural frequency of the vibration of a tapered bar fixed at its base using Rayleigh-Ritz method as shown in figure. Take the width of the bar as unity.	20	C02 C04
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The diagram shows a tapered bar fixed at its base on the left. The bar tapers from a larger width at the base to a smaller width at the free end on the right. The total length of the bar is  $L$ . A coordinate  $x$  is shown along the length of the bar, and a differential element  $dx$  is indicated.

	OR		
Q7	An automobile is modeled with a capability of pitch and bounce motions, as shown in Fig. It travels on a rough road whose surface varies sinusoidally with an amplitude of 0.035 m and a wavelength of 7.5 m. Derive the equations of motion of the automobile for the following data: Radius of gyration = 1.2 m Velocity = 50 km/hr. Location of CG from front axle = 1015 mm Location of CG from rear axle = 1240 mm	20	C02 C04

Stiffness of front tire and suspension = 20 kN/m  
Stiffness of rear tire and suspension = 16 kN/m  
Also calculate the pitching and bouncing frequency of the car in motion

